

INPUT VALUES, Nozzle Description: BOOT From : 20

Pressure for Reinforcement Calculations	P	11.016	kgf/cm ²
Temperature for Internal Pressure	Temp	310	C
Design External Pressure	Pext	2.40	kgf/cm ²
Temperature for External Pressure	Tempex	175	C
Maximum Allowable Pressure New & Cold		16.21	kgf/cm ²
Shell Material [Normalized]		SA-516	70
Shell Allowable Stress at Temperature	S	1368.19	kgf/cm ²
Shell Allowable Stress At Ambient	Sa	1406.14	kgf/cm ²
Inside Diameter of Cylindrical Shell	D	2600.00	mm
Design Length of Section	L	2345.0002	mm
Shell Finished (Minimum) Thickness	t	26.0000	mm
Shell Internal Corrosion Allowance	c	3.0000	mm
Shell External Corrosion Allowance	co	1.5000	mm
Physical Maximum for Diameter Limit Dmax		2000.0000	mm
Physical Maximum for Thickness Limit Tmax		0.0000	mm
Distance from Bottom/Left Tangent		3390.0000	mm
User Entered Minimum Design Metal Temperature		11.00	C

Type of Element Connected to the Shell : Nozzle

Material [Normalized]	SA-105
Material UNS Number	K03504
Material Specification/Type	Forgings

Allowable Stress at Temperature	Sn	1302.13	kgf/cm ²
Allowable Stress At Ambient	Sna	1406.14	kgf/cm ²
Diameter Basis (for tr calc only)	ID		
Layout Angle		180.00	deg
Diameter		47.2400	in.
Size and Thickness Basis	Actual		
Actual Thickness	tn	16.0000	mm
Hub Height of Integral Nozzle	h	80.0000	mm
Height of Beveled Transition	L`	100.0000	mm
Hub Thickness of Integral Nozzle (tn or x+tp)		80.0000	mm
Corrosion Allowance	can	7.5000	mm
Joint Efficiency of Shell Seam at Nozzle	E1	1.00	
Joint Efficiency of Nozzle Neck	En	1.00	
Outside Projection	ho	2655.0000	mm
Weld leg size between Nozzle and Pad/Shell	Wo	16.0000	mm
Groove weld depth between Nozzle and Vessel	Wgnv	26.0000	mm
Inside Projection	h	0.0000	mm
Weld leg size, Inside Element to Shell	Wi	0.0000	mm
ASME Code Weld Type per UW-16		None	

The Pressure Design option was MAWP + static head (to the nozzle).

Nozzle Sketch (may not represent actual weld type/configuration)

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Hub Nozzle

Reinforcement CALCULATION, Description: BOOT

ASME Code, Section VIII, Division 1, 2007, A-09 UG-37 to UG-45

Actual Inside Diameter Used in Calculation 47.240 in.

Actual Thickness Used in Calculation 0.630 in.

Nozzle input data check completed without errors.

Reqd thk per UG-37(a) of Cylindrical Shell, Tr [Int. Press]

$$\begin{aligned}
 &= (P \cdot R) / (S \cdot E - 0.6 \cdot P) \text{ per UG-27 (c) (1)} \\
 &= (11.02 \cdot 1303.0001) / (1368 \cdot 1.00 - 0.6 \cdot 11.02) \\
 &= 10.5422 \text{ mm}
 \end{aligned}$$

Reqd thk per UG-37(a) of Cylindrical Shell, Tr [Mapnc]

$$\begin{aligned}
 &= (P \cdot R) / (S \cdot E - 0.6 \cdot P) \text{ per UG-27 (c) (1)} \\
 &= (16.21 \cdot 1300.0001) / (1406 \cdot 1.00 - 0.6 \cdot 16.21) \\
 &= 15.0870 \text{ mm}
 \end{aligned}$$

Reqd thk per UG-37(a) of Nozzle Wall, Trn [Int. Press]

$$= (P \cdot R) / (S \cdot E - 0.6 \cdot P) \text{ per UG-27 (c) (1)}$$

$$= (11.02 \cdot 607.45) / (1302 \cdot 1.00 - 0.6 \cdot 11.02)$$

$$= 5.1653 \text{ mm}$$

Reqd thk per UG-37(a) of Nozzle Wall, Trn [Mapnc]

$$= (P \cdot R) / (S \cdot E - 0.6 \cdot P) \text{ per UG-27 (c) (1)}$$

$$= (16.21 \cdot 599.95) / (1406 \cdot 1.00 - 0.6 \cdot 16.21)$$

$$= 6.9626 \text{ mm}$$

Required Nozzle thickness under External Pressure per UG-28 : 7.5360 mm

Intermediate Hub Nozzle Calculations:

Check to determine use of Sketch (e-1) or (e-2):

Height value from sketch (e-1) [te]:

$$= (\text{Hub Thickness} - \text{Neck Thickness}) / \cos(30)$$

$$= 64.000 / 0.5773$$

$$= 110.8513 \text{ mm}$$

Note: Hub Height was < 2.5 times Hub Thickness, use sketch UG-40 (e-1).

UG-40, Limits of Reinforcement : [Int. Press]

Parallel to Vessel Wall (Diameter Limit)	D1	2000.0000	mm
Parallel to Vessel Wall, opening length	d	1000.0000	mm
Normal to Vessel Wall (Thickness Limit), no pad	Tlnp	53.7500	mm

Intermediate Hub Nozzle Calculations:

Check to determine use of Sketch (e-1) or (e-2):

Height value from sketch (e-1) [te]:

$$\begin{aligned}
 &= (\text{Hub Thickness} - \text{Neck Thickness}) / \cos(30) \\
 &= 64.000 / 0.5773 \\
 &= 110.8513 \text{ mm}
 \end{aligned}$$

Note: Hub Height was < 2.5 times Hub Thickness, use sketch UG-40 (e-1).

UG-40, Limits of Reinforcement : [Mapnc]

Parallel to Vessel Wall (Diameter Limit)	D1	2000.0000	mm
Parallel to Vessel Wall, opening length	d	1000.0000	mm
Normal to Vessel Wall (Thickness Limit), no pad	Tlnp	65.0000	mm

Weld Strength Reduction Factor [fr1]:

$$\begin{aligned}
 &= \min(1, S_n/S) \\
 &= \min(1, 1302.1 / 1368.2) \\
 &= 0.952
 \end{aligned}$$

Weld Strength Reduction Factor [fr2]:

$$\begin{aligned}
 &= \min(1, S_n/S) \\
 &= \min(1, 1302.1 / 1368.2) \\
 &= 0.952
 \end{aligned}$$

Weld Strength Reduction Factor [fr3]:

$$\begin{aligned}
 &= \min(fr2, fr4) \\
 &= \min(1.0 , 1.0) \\
 &= 0.952
 \end{aligned}$$

Results of Nozzle Reinforcement Area Calculations:

AREA AVAILABLE, A1 to A5	MAWP	External	Mapnc	
Area Required	Ar	128.815	68.572	181.029 cm ²

Area in Shell	A1	85.263	79.959	87.315	cm ²
Area in Nozzle Wall	A2	3.412	0.986	11.749	cm ²
Area in Inward Nozzle	A3	0.000	0.000	0.000	cm ²
Area in Welds	A41+A42+A43	1.833	1.833	1.926	cm ²
Area in Element	A5	0.000	0.000	0.000	cm ²
Area in Hub	A6	65.478	65.478	83.200	cm ²
TOTAL AREA AVAILABLE	Atot	155.986	148.257	184.190	cm ²

The MAP(nc) Case Governs the Analysis.

Nozzle Angle Used in Area Calculations 90.00 Degs.

The area available without a pad is Sufficient.

Area Required [A]:

$$\begin{aligned}
 &= (d * tr * F + 2 * tn * tr * F * (1 - fr1)) \text{ UG-37(c)} \\
 &= (1199.8960 * 15.0870 * 1.0 + 2 * 80.0000 * 15.0870 * 1.0 * (1 - 1.00)) \\
 &= 181.029 \text{ cm}^2
 \end{aligned}$$

Reinforcement Areas per Figure UG-37.1

Area Available in Shell [A1]:

$$\begin{aligned}
 &= d(E1 * t - F * tr) - 2 * tn(E1 * t - F * tr) * (1 - fr1) \\
 &= 800.104 (1.00 * 26.0000 - 1.0 * 15.087) - 2 * 80.000 \\
 &\quad (1.00 * 26.0000 - 1.0 * 15.0870) * (1 - 1.000) \\
 &= 87.315 \text{ cm}^2
 \end{aligned}$$

Area Available in Nozzle Projecting Outward [A2]:

$$\begin{aligned}
 &= (2 * tlnp) * (tn - trn) * fr2 \\
 &= (2 * 65.00) * (16.00 - 6.96) * 1.0000) \\
 &= 11.749 \text{ cm}^2
 \end{aligned}$$

Area Available in Inward Weld + Outward Weld [A41 + A43]:

$$\begin{aligned}
 &= W_o^2 * fr2 + (W_i - can / 0.707)^2 * fr2 \\
 &= 13.8784^2 * 1.0000 + (0.0000)^2 * 1.0000 \\
 &= 1.926 \text{ cm}^2
 \end{aligned}$$

Area Available in the Hub Section [A6]:

$$\begin{aligned}
 &= (2 * \min(T_{lnp}, h_o, Hubht)) * (Hubtk - t_n) * fr2 \\
 &= (2 * \min(53.8, 2655.0, 80.0)) * (80.0 - 16.0) * 1.000 \\
 &= 83.200 \text{ cm}^2
 \end{aligned}$$

UG-45 Minimum Nozzle Neck Thickness Requirement: [Int. Press.]

$$\begin{aligned}
 \text{Wall Thickness per UG45(a),} \quad tra &= 15.0360 \text{ mm} \\
 \text{Wall Thickness per UG16(b),} \quad tr16b &= 9.0000 \text{ mm} \\
 \text{Wall Thickness per UG45(b)(1),} \quad trb1 &= 15.0422 \text{ mm} \\
 \text{Wall Thickness per UG45(b)(2),} \quad trb2 &= 6.7880 \text{ mm} \\
 \text{Wall Thickness per UG45(b)(3),} \quad trb3 &= \text{Max}(trb1, trb2, tr16b) = 15.0422 \text{ mm} \\
 \text{Std. Wall Pipe per UG45(b)(4),} \quad trb4 &= 15.8344 \text{ mm} \\
 \text{Wall Thickness per UG45(b),} \quad trb &= \text{Min}(trb3, trb4) = 15.0422 \text{ mm} \\
 \\
 \text{Final Required Thickness,} \quad tr45 &= \text{Max}(tra, trb) = 15.0422 \text{ mm} \\
 \text{Available Nozzle Neck Thickness} &= 16.0000 \text{ mm} \rightarrow \text{OK}
 \end{aligned}$$

UG-45 Minimum Nozzle Neck Thickness Requirement: [MAPnc]

$$\begin{aligned}
 \text{Wall Thickness per UG45(a),} \quad tra &= 6.9626 \text{ mm} \\
 \text{Wall Thickness per UG16(b),} \quad tr16b &= 1.5000 \text{ mm} \\
 \text{Wall Thickness per UG45(b)(1),} \quad trb1 &= 15.0870 \text{ mm} \\
 \text{Wall Thickness per UG45(b)(3),} \quad trb3 &= \text{Max}(trb1, trb2, tr16b) = 15.0870 \text{ mm} \\
 \text{Std. Wall Pipe per UG45(b)(4),} \quad trb4 &= 8.3344 \text{ mm} \\
 \text{Wall Thickness per UG45(b),} \quad trb &= \text{Min}(trb3, trb4) = 8.3344 \text{ mm}
 \end{aligned}$$

Final Required Thickness, $tr_{45} = \text{Max}(tr_a, tr_b) = 8.3344 \text{ mm}$

Available Nozzle Neck Thickness = 16.0000 mm --> OK

Nozzle Junction Minimum Design Metal Temperature (MDMT) Calculations:

MDMT of Nozzle-Shell/Head Weld for the Nozzle (UCS-66(a)1(b)), **Curve: C**

Govrn. thk, $t_g = 26.000$, $t_r = 10.256$, $c = 4.5000 \text{ mm}$, $E^* = 1.00$

Stress Ratio = $t_r * (E^*) / (t_g - c) = 0.477$, Temp. Reduction = 38 C

Min Metal Temp. w/o impact per UCS-66 -20 C

Min Metal Temp. at Required thickness (UCS 66.1) -48 C

Governing MDMT of all the sub-joints of this Junction : -48 C

Large Opening Calculations per Appendix 1-10

Thickness of Nozzle at Shell [tn]:

= hub thickness - corrosion allowance

= 80.000 - 7.500

= 72.500 mm

Thickness of Nozzle at Top [tn2]:

= thickness - corrosion allowance

= 16.000 - 7.500

= 8.500 mm

Effective Pressure Radius [Reff]:

= $D_i/2$ + corrosion allowance

= 2600.000 /2 + 3.000

$$= 1303.000 \text{ mm}$$

Effective Length of Vessel Wall [LR]:

$$= 8 * t$$

$$= 8 * 21.500$$

$$= 172.000 \text{ mm}$$

Thickness Limit Candidate [LH1]:

$$= t + 0.78 * \sqrt{R_n * t_n}$$

$$= 21.500 + 0.78 * \sqrt{607.448 * 72.500}$$

$$= 185.188 \text{ mm}$$

Thickness Limit Candidate [LH2]:

$$= L_{pr1} + t$$

$$= 2655.000 + 21.500$$

$$= 2676.500 \text{ mm}$$

Thickness Limit Candidate [LH3]:

$$= 8 * (t + t_e)$$

$$= 8 * (21.500 + 0.000)$$

$$= 172.000 \text{ mm}$$

Thickness Limit Candidate [LH4]:

$$= L_{pr3} + t + 0.78 * (T_{N2} / T_N) * \sqrt{r_n * t_{n2}}$$

$$= 80.00 + 21.50 + 0.78 * (8.50 / 72.50) * \sqrt{607.45 * 8.50}$$

$$= 108.071 \text{ mm}$$

Effective Nozzle Wall Length Outside the Vessel [LH]:

$$= \min[L_{H1}, L_{H2}, L_{H4}]$$

$$= \min[185.188, 2676.500, 108.071]$$

$$= 108.071 \text{ mm}$$

Effective Vessel Thickness [teff]:

$$= t$$

$$= 21.500 \text{ mm}$$

Determine Parameter [Lamda]:

$$= \min(10, (D_n + T_n) / (\sqrt{ (D_i + t_{eff}) * t_{eff} }))$$

$$= \min(10, (1214.90 + 72.500) / (\sqrt{ (2606.00 + 21.500) * 21.500 }))$$

$$= 5.417$$

Compute Areas A1-A43 (No Pad) or A1-A5 (With Pad) :

Area Contributed by the Vessel Wall [A1]:

$$= t * L_R * \max(\text{Lamda}/4, 1)$$

$$= 21.500 * 172.000 * \max(5.417 / 4, 1)$$

$$= 50.076 \text{ cm}^2$$

Area Contributed by the Nozzle Outside the Vessel Wall [A2]:

$$= t_n * (L_{pr3} + t) + 0.78 * (t_n^2 / t_n) * \sqrt{ R_n * t_n^2 }$$

$$= 72.500 * (80.000 + 21.500) + 0.78 * (8.500^2 / 72.500) * \sqrt{ 6 }$$

$$= 74.146 \text{ cm}^2$$

Area Contributed by the Outside Fillet Weld [A41]:

$$= 0.5 * \text{Leg41}^2$$

$$= 0.5 * 16.000^2$$

$$= 1.280 \text{ cm}^2$$

The total area contributed by A1 through A43 [AT]:

$$= A1 + \text{frn}(A2 + A3) + A41 + A42 + A43$$

$$= 50.076 + 1.000(74.146 + 0.000) + 1.280 + 0.000 + 0.000$$

$$= 125.502 \text{ cm}^2$$

Allowable Local Primary Membrane Stress [Sallow]:

$$\begin{aligned}
 &= 1.5 * S * E \\
 &= 1.5 * 1368.194 * 1.000 \\
 &= 2052.3 \text{ kgf/cm}^2
 \end{aligned}$$

Determine Force acting on the Nozzle [fN]:

$$\begin{aligned}
 &= P * R_n * (L_H - t) \\
 &= 11.016 * 607.448 * (108.071 - 21.500) \\
 &= 5793.1 \text{ kgf}
 \end{aligned}$$

Determine Force acting on the Shell [fS]:

$$\begin{aligned}
 &= P * R_{eff} * (L_R + t_n) \\
 &= 11.016 * 1303.000 * (172.000 + 72.500) \\
 &= 35095.7 \text{ kgf}
 \end{aligned}$$

Discontinuity Force from Internal Pressure [fY]:

$$\begin{aligned}
 &= P * R_{eff} * R_{nc} \\
 &= 11.016 * 1303.000 * 607.448 \\
 &= 87193.5 \text{ kgf}
 \end{aligned}$$

Area Resisting Internal Pressure [Ap]:

$$\begin{aligned}
 &= R_n (L_H - t) + R_{eff} (L_R + t_n + R_{nc}) \\
 &= 607.448 (108.071 - 21.500) + 1303.000 (172.000 + 72.500 + 607.448) \\
 &= 11626.8 \text{ cm}^2
 \end{aligned}$$

Maximum Allowable Working Pressure Candidate [Pmax1]:

$$\begin{aligned}
 &= S_{allow} / (2 * A_p / A_T - R_{xs} / t_{eff}) \\
 &= 2052.291 / (2 * 11626.758 / 125.502 - 1303.000 / 21.500) \\
 &= 16.5 \text{ kgf/cm}^2
 \end{aligned}$$

Maximum Allowable Working Pressure Candidate [Pmax2]:

$$\begin{aligned}
 &= S[t/Reff] \\
 &= 1368.194 [21.500 / 1303.000] \\
 &= 22.6 \text{ kgf/cm}^2
 \end{aligned}$$

Maximum Allowable Working Pressure [Pmax]:

$$\begin{aligned}
 &= \min(Pmax1, Pmax2) \\
 &= \min(16.461 , 22.576) \\
 &= 16.461 \text{ kgf/cm}^2
 \end{aligned}$$

Average Primary Membrane Stress [SigmaAvg]:

$$\begin{aligned}
 &= (fN + fS + fY) / AT \\
 &= (5793.119 + 35095.676 + 87193.453) / 125.502 \\
 &= 1020.562 \text{ kgf/cm}^2
 \end{aligned}$$

General Primary Membrane Stress [SigmaCirc]:

$$\begin{aligned}
 &= P * Reff / teff \\
 &= 11.016 * 1303.000 / 21.500 \\
 &= 667.6 \text{ kgf/cm}^2
 \end{aligned}$$

Maximum Local Primary Membrane Stress [PL]:

$$\begin{aligned}
 &= \max(2 * SigmaAvg - SigmaCirc, SigmaCirc) \\
 &= \max(2 * 1020.562 - 667.632 , 667.632) \\
 &= 1373.5 \text{ kgf/cm}^2
 \end{aligned}$$

Summary of Nozzle Pressure/Stress Results:

Allowed Local Primary Membrane Stress	Sallow	2052.29	kgf/cm ²
Local Primary Membrane Stress	PL	1373.49	kgf/cm ²
Maximum Allowable Working Pressure	Pmax	16.46	kgf/cm ²

Weld Size Calculations, Description: BOOT (HOLD)

Intermediate Calc. for nozzle/shell Welds Tmin 19.0000 mm

Results Per UW-16.1:

	Required Thickness	Actual Thickness
Nozzle Weld	6.0000 = Min per Code	11.3120 = 0.7 * Wo mm

Weld Strength and Weld Loads per UG-41.1, Sketch (a) or (b)

Weld Load [W]:

$$\begin{aligned}
 &= (A - A_1 + 2 * t_n * f_{r1} * (E_1 * t - t_r)) * S_v \\
 &= (181.0288 - 87.3150 + 2 * 80.0000 * 0.9517 * \\
 &\quad (1.00 * 24.5000 - 15.0870)) * 1368 \\
 &= 147829.47 \text{ kgf}
 \end{aligned}$$

For hub type nozzles, A2 includes the area area of the hub.

Note: F is always set to 1.0 throughout the calculation.

Weld Load [W1]:

$$\begin{aligned}
 &= (A_2 + A_4 - (W_i - C_{an} / .707)^2 * f_{r2}) * S_v \\
 &= (94.9486 + 1.9261 - 0.0000 * 0.95) * 1368 \\
 &= 132543.14 \text{ kgf}
 \end{aligned}$$

Weld Load [W2]:

$$\begin{aligned}
 &= (A_2 + A_3 + A_4 + (2 * t_n * t * f_{r1})) * S_v \\
 &= (94.9486 + 0.0000 + 1.9261 + (37.3072)) * 1368 \\
 &= 183586.56 \text{ kgf}
 \end{aligned}$$

Weld Load [W3]:

$$\begin{aligned}
 &= (A_2 + A_3 + A_4 + A_5 + (2 * t_n * t * f_{r1})) * S \\
 &= (94.9486 + 0.0000 + 1.9261 + 0.0000 + (37.3072)) * 1368
 \end{aligned}$$

$$= 183586.56 \text{ kgf}$$

Strength of Connection Elements for Failure Path Analysis

Shear, Outward Nozzle Weld [Sonw]:

$$\begin{aligned} &= (\pi/2) * D_{lo} * W_o * 0.49 * S_{nw} \\ &= (3.1416 / 2.0) * 1359.8960 * 16.0000 * 0.49 * 1302 \\ &= 218070. \text{ kgf} \end{aligned}$$

Shear, Nozzle Wall [Snw]:

$$\begin{aligned} &= (\pi * (D_{lr} + D_{lo}) / 4) * (Thk - Can) * 0.7 * S_n \\ &= (3.1416 * 639.9481) * (80.0000 - 0.0000) * 0.7 * 1302 \\ &= 1466008. \text{ kgf} \end{aligned}$$

Tension, Shell Groove Weld [Tngw]:

$$\begin{aligned} &= (\pi/2) * D_{lo} * (W_{gnvi} - Cas) * 0.74 * S_{ng} \\ &= (3.1416 / 2.0) * 1359.8960 * (26.0000 - 1.5000) * 0.74 * 1368 \\ &= 529871. \text{ kgf} \end{aligned}$$

Strength of Failure Paths:

$$\begin{aligned} \text{PATH11} &= (\text{SONW} + \text{SNW}) = (218069 + 1466008) = 1684077 \text{ kgf} \\ \text{PATH22} &= (\text{Sonw} + \text{Tpgw} + \text{Tngw} + \text{Sinw}) \\ &= (218069 + 0 + 529871 + 0) = 747941 \text{ kgf} \\ \text{PATH33} &= (\text{Sonw} + \text{Tngw} + \text{Sinw}) \\ &= (218069 + 529871 + 0) = 747941 \text{ kgf} \end{aligned}$$

Summary of Failure Path Calculations:

Path 1-1 = 1684077 kgf, must exceed W = 147829 kgf or W1 = 132543 kgf
 Path 2-2 = 747941 kgf, must exceed W = 147829 kgf or W2 = 183586 kgf
 Path 3-3 = 747941 kgf, must exceed W = 147829 kgf or W3 = 183586 kgf

Maximum Allowable Pressure for this Nozzle at this Location:

Converged Max. Allow. Pressure in Operating case	11.782	kgf/cm ²
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Note: The MAWP of this junction was limited by the parent Shell/Head.

Nozzle is O.K. for the External Pressure	2.400	kgf/cm ²
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Approximate M.A.P.(NC) for given geometry	16.375	kgf/cm ²
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The Drop for this Nozzle is : 191.9970 mm

The Cut Length for this Nozzle is, Drop + Ho + H + T : 2872.9968 mm

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