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## AUTHOR

DEPARTMENT

LOCATION
DATE
B. Ghiorso

Mech. Engr.
BERKELEY
01/23/06
PROGRAM-PROJECT-JOB

## US - LHC COLLABORATION

## LUMI PRESSURE VESSEL DESIGN SAFETY NOTE

## Design Features:

$>\quad$ Vessel is designed to be used at up to 132 psi using a mix of $96 \%$ argon, $4 \%$ nitrogen (inert gas) with a total stored energy of less than 5000 joules without contents, less that 2600 joules with its contents, thereby placing it in the low-hazard pressure systems category (see section 7.5.1 of PUB-3000 Rev'd 1/99)
$>$ Constructed from EDM'ed solid Type 304 stainless steel billet with resulting wall thicknesses of 0.188 inch [4.7mm] and 0.125 inch [3mm]
$>$ Tank maximum stress is 15740 psi safety factor on ultimate strength is 4.8 (Type 304 stainless steel tensile strength $=75400 \mathrm{psi}$ ).
$>$ Top and bottom vessel components are machined with corner transition regions so welds are removed from high stress areas [see drawing] and have less than 6300 psi stress.
$>\quad$ Vessel cover plate fastener stress is: $10.6 \mathrm{in}^{2} \times 14.7 \mathrm{psi} \times 9 \mathrm{~atm} . / 24$ fasteners $/ .01948 \mathrm{in}^{2}($ area of fastener $)=3000 \mathrm{psi}$
$>$ Tank exterior is plasma coated with alumina (coating thickness aproximately 0.004 inches) prior to hydrostatic testing.
$>$ All units are hydrostatically tested at 1.5 x design pressure and are stamped: "TESTED: MAX PRESSURE 132 PSI" $<$ PRESSURE TEST NO. $>$ on side of tank flange. Test data is appended to and filed as a revision of this engineering note.
> Pressure delivery system is limited to 132 psi by an overpressure safety valve in gas delivery system.
$>\quad$ Vessel will normally be used in an un-manned area.
Refer to the following pages:
2) Mechanical drawing
3) Pre-weld assembly exploded view showing weld seam locations
4) Stored energy calculation

5-7) Stress plots showing stress at weld seam locations as well as maximum stress in model

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INSTANCE: CASE_WELD_PA
Pressure vessel components

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Appendix E (LBNL PUB 3000)
STORED ENERGY OF A PRESSURED GAS VESSEL
Although the pressure section of this publication is not intended to be a primer on pressure calculation, the following formula is used sufficiently frequently, but is obscure enough, that it has been included.

When a gas is compressed, energy is stored in it. If the energy is released in an unfavorable way, it will cause damage. Stored energies in excess of 100 kJ are considered high hazard. Sometimes it is helpful to think of stored energy in terms of grams of TNT. One gram of TNT contains 4.62 kJ of energy.

Volume of vessel: $\quad \mathrm{Vh}:=283 \cdot \mathrm{in}^{3}$
Absolute pressure of vessel: $\quad \mathrm{Ph}:=147 \cdot \mathrm{psi}$
Pressure to which vessel would drop if burst:

> P1 := 14.7•psi

$$
\begin{aligned}
& \mathrm{Vh}=0.004638 \mathrm{~m}^{3} \\
& \mathrm{Ph}=1.01353 \times 10^{6} \mathrm{~Pa} \\
& \mathrm{P} 1=1.01353 \times 10^{5} \mathrm{~Pa} \\
& \gamma:=1.66667 \quad \text { (argon) }
\end{aligned}
$$

$\gamma=$ The adiabatic exponent or ratio of specific
heats, $\mathrm{Cp} / \mathrm{Cv}$. The value is 1.666 for monatomic gases such as argon and helium; 1.4 for diatomic gases such as nitrogen, oxygen, hydrogen, and air; and variable for polyatomic gases such as methane, water, and carbon dioxide but generally very nearly 1.3.

$$
\frac{\mathrm{Ph} \cdot \mathrm{Vh}}{\gamma-1} \cdot\left[1-\left(\frac{\mathrm{P} 1}{\mathrm{Ph}}\right)^{\frac{\gamma-1}{\gamma}}\right]=4243.59 \mathrm{~N} \cdot \mathrm{~m} \quad \text { (joules) }
$$

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Pressure vessel top at 132 psi showing weld seam stress < 6300 psi


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psi

```
1.417e+04
1.260e+04
1.103e+\square4
9.453e+03
7.880e+03
6.307e+03
4.735e+03
3.162e+03
1.590e+03
```

Pressure vessel interior view at 132 psi showing maximum stress < 16000 psi

